

AMENDMENTS TO THE SPECIFICATION

Please replace page 2, lines 7-24 with the following paragraph rewritten in amendment format:

For example, in a refrigerant evaporator shown in Fig. 19, a plurality of flat tubes 120 are layered between an upper tank 116 and a lower tank 118. The tubes 120 forms a core portion 122. A refrigerant inlet connector 112 and a refrigerant outlet connector 114 are connected to a left end and a right end of the upper tank 116. A separator ~~[[24]]~~ 124 is provided in a middle portion of the upper tank ~~[[16]]~~ 116. The refrigerant flows in the left tubes ~~[[20]]~~ 120, which are arranged in a left section of the core portion ~~[[22]]~~ 122, at the substantially same time and makes a turn in the lower tank 118 from the left side to the right side. Then, the refrigerant flows in the right tubes 120, which are arranged in a right section of the core portion 122. Thus, a refrigerant first pass ~~[[T1]]~~ P1 is made in the left section and a refrigerant second pass ~~[[T2]]~~ P2 is made in the right section, when viewed in a broad aspect. Here, even if the refrigerant evaporator is placed such that the upper tank 116 and the lower tank 118 extend vertically and the tubes 120 are layered in a vertical direction, the direction that the tubes 120 are layered is still referred to as the core width direction D1.

Please replace page 3, lines 4-10 with the following paragraph rewritten in amendment format:

Also in a case that the refrigerant does not have super heat, it is necessary to uniformly distribute the liquid refrigerant in the right tubes 120 because the amount of the refrigerant is generally small. If the refrigerant is not uniformly distributed in the

tubes ~~[[20]]~~ 120, the refrigerant will be dried out, that is, completely evaporated in the tubes ~~[[20]]~~ 120 in which the amount of the refrigerant is small. As a result, the temperature of air is not uniform.

Please replace page 3, lines 11-27 with the following paragraph rewritten in amendment format:

To solve this problem, a 2-2 pass-type evaporator shown in Figs. 20A, 20B is proposed. It is for example disclosed in US 6,272,881B1 (JP-A-11-287587). In the 2-2 pass-type evaporator, a front core portion 122A and a rear core portion 122B are arranged between a pair of upper tanks 116A, 116B and a pair of lower tanks 118A, 118B. A refrigerant inlet and outlet connector 113 is connected to a upper left end of the upper tanks 116A, ~~[[6B]]~~ 116B. A separator 124A is provided in the upper front tank 116A, which communicates with the refrigerant inlet and a separator 124B is provided in the upper rear tank 116B, which communicates with the refrigerant outlet. Thus, ~~[[to]]~~ two refrigerant passes P1 and P2 are made in the front core portion 122A and two refrigerant passes P3 and P4 are made in the rear core portion 122B, from a broad view. As shown in Fig. 20B, the front core portion 122A is constructed of a row of tubes 120A and the rear core portion 122B is constructed of a row of tubes 120B. Corrugated fins 126 are interposed between the tubes 120A, 120B.

Please replace page 4, lines 1-8 with the following paragraph rewritten in amendment format:

In the above evaporator, since the refrigerant flows through four passes P1 to P4, the flow distance of the refrigerant is long. Also, the refrigerant turns many times. That is, the numbers that the refrigerant flows in and out the tubes [[20A]] 120A, [[20B]] 120B and the core portions [[22A]] 122A, [[22B]] 122B is increased (four times in Fig. 20A). Therefore, the pressure loss of the refrigerant is increased throughout the evaporator. As a result, the performance of the evaporator is deteriorated.